Heat Transfer (1)

Modes of Heat transfer

15/3/2016

AMR

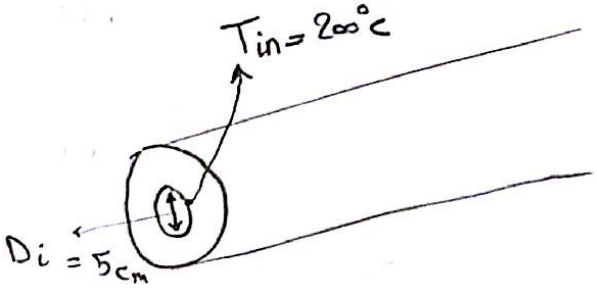
sheet . No.1

Prob (16) Pipe surrounded by asbestos.

$K_{asb} = 0.17\ W/m^\circ C$

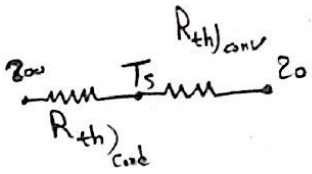
$T_\infty = 20^\circ C$

$h_o = 3\ W/m^2 . ^\circ C$



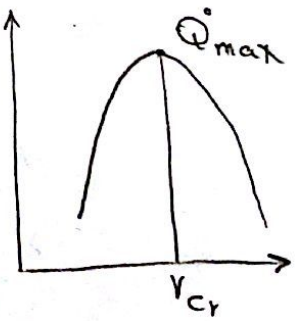
Require

1. $r_{cr} = ?$ mm
2. $\dot{Q}_{loss}$ without insulation.
3. $\dot{Q}_{loss}$ pipe covered with critical radius of insulation.



Solution

* Cylinder (pipe) tube $\longrightarrow$ $r_{cr} = \frac{K_{ins}}{h_o} = \frac{0.17}{3} = 0.056\ m = 5.6\ cm$



$$\dot{Q}_{with\ insulation} = \frac{\Delta T}{\Sigma R_{th}} = \frac{200 - 20}{\frac{\ln r_{cr}/r_i}{2\pi K_{ins} L} + \frac{1}{h_* 2\pi r_c * L}}$$

$$q' = \frac{\dot{Q}}{L} = \frac{180}{\frac{\ln 5.6/2.5}{2\pi * 0.17 * 1} + \frac{1}{3 * 2\pi * 0.056}} = 105.73\ \frac{W}{m}$$

Case [2]

$$Q^{\circ}_{\text{without insulation}} = \frac{\Delta T}{R_{th)_{conv}}} = \frac{200 - 20}{\frac{1}{3 * 2\pi * 0.025}} = 84.82 \ W/m.$$

Prob [17] steel pipe ($K_s = 43 \ W/m.^{\circ}c$)

$D_{o\,pipe} = 2.5 \ cm = 25 \ mm$

wall thickness = 2 mm.

$h_i = 500 \ W/m^2.^{\circ}c$

$h_o = 12 \ W/m^2.^{\circ}c$

Calculate "U" the overall heat transfer coefficient.

* IF the pipe is covered with asbestos.

$K_{asb} = 0.18 \ W/m.^{\circ}c$ and $h_o = 12 \ W/m^2.^{\circ}c$

Determine [1] r_{cr}

[2] will the heat transfer be increased or decreased by adding an insulation thickness [a] 5 mm.

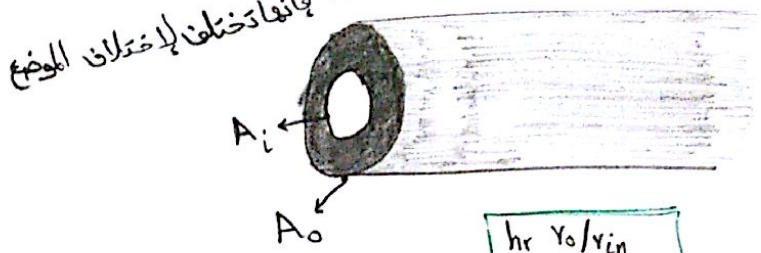
[b] 10 mm.

over all heat transfer coefficient "U"

$$\therefore Q^\circ = \frac{\Delta T}{\Sigma R_{th}} = U * A * \Delta T$$

$$\therefore UA = \frac{1}{\Sigma R_{th}} \qquad \therefore U = \frac{1}{A \cdot \Sigma R_{th}}$$

تعتمد على (A)
لذا فإنها تختلف لإختلاف الموضع

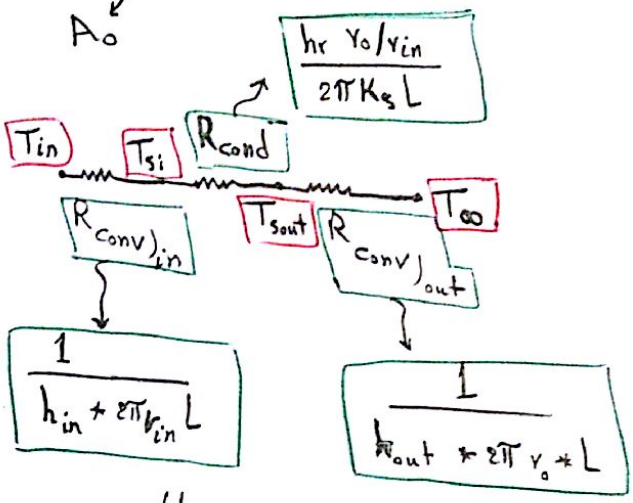


$D_o = 2.5\ cm = 25\ mm$

$r_o = 12.5\ mm$

thickness wall = 2 mm.

$r_{in} = 12.5 - 2 = 10\ mm$



$$\Sigma R_{th} = \frac{1}{h_i * 2\pi r_i L} + \frac{\ln r_o/r_i}{2\pi K_s L} + \frac{1}{h_o * 2\pi r_o * L} = \text{؟؟}$$

$$U_i = \frac{1}{A_i * \Sigma R_{th}} = \text{؟؟} \quad \& \quad U_o = \frac{1}{A_o * \Sigma R_{th}} = \text{؟؟}$$

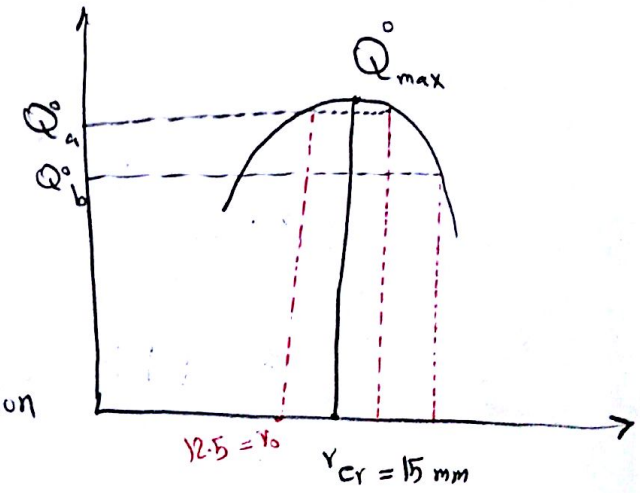
$$r_{cr} = \frac{K_{asb}}{h_o} = \frac{0.18}{12} = 0.015\ m = \underline{15}\ mm$$

[a] thickness $\delta_{insulation} = 5\ mm$

(d)

$r_o = 12.5$ mm

$r_{insulation)_a} = 12.5 + 5 = 17.5$ mm

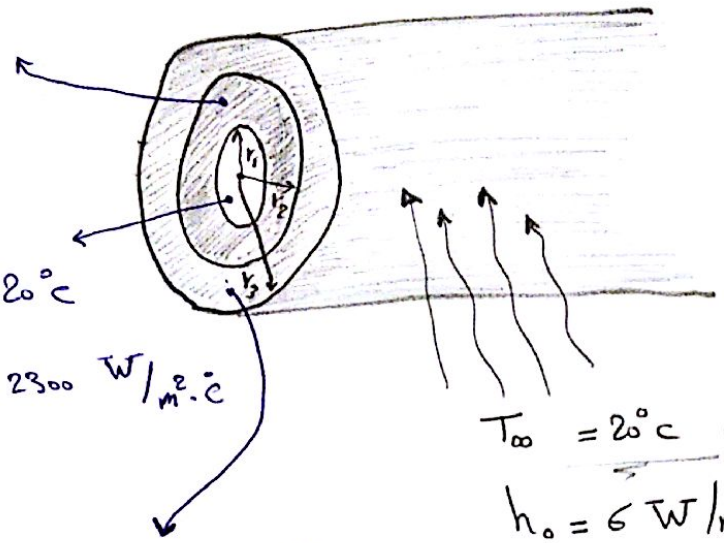


$Q^\circ_a \equiv$ constant and the insulation not effective

$r_{insulation)_b} = 12.5 + 10 = 22.5$ mm

$Q^\circ_b \longrightarrow$ is decreased and the insulation will be effective.

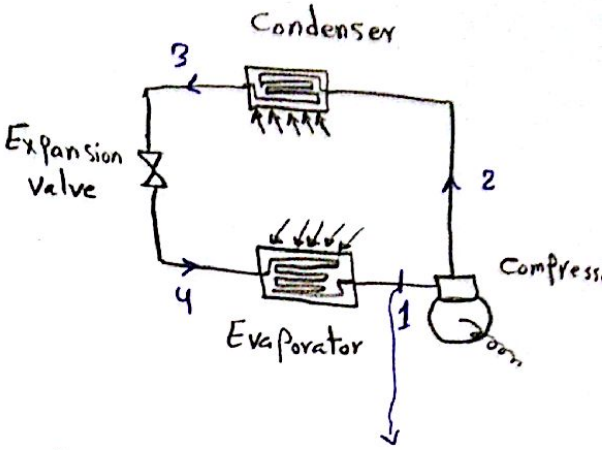
$k_p = 58\ W/m.^\circ C$, $D_i = 50\ mm$, $\delta_w = 5mm$



$T_R = -20^\circ C$

$h_R = 2300\ W/m^2.^\circ C$

$k_{in} = 0.042\ W/m^2.^\circ C$



suction line
خط السحب

$T_\infty = 20^\circ C$.

$h_o = 6\ W/m^2.^\circ C$.

Dew Point Temperature (درجة حرارة قطرة الندى)

هي درجة الحرارة التي عندها جزء من بخار الماء يتكثف

$T_{dp} = 15^\circ C$

Required: δ insulation to prevent water vapour From condensed on the outer surface of tube.

Sol..

$r_3 - r_2 = \delta_{insulation}$

For steady state Flow of heat

$$\dot{Q}_{air} = \dot{Q}_{outer\ surface}$$

$\dot{Q}_{air}$ → outer surface

$\dot{Q}_{outer\ surface}$ → Refrigerant

$$\frac{T_\infty - T_{s\,out}}{\frac{1}{h_o * 2\pi r_3 * L}} = \frac{T_{s\,out} - (-20)}{\frac{\ln r_3/r_2}{2\pi k_{ins} * L} + \frac{\ln r_2/r_1}{2\pi k_{pipe} * L} + \frac{1}{h_i * 2\pi r_i * L}}$$

F

let $T_{s\,at} = 16^{\circ}c$

by using (Trial and error)

Let $r_3 = 32$ mm ⟶ R.H.S = VV ? L.H.S = VV ✗

$r_3 = 38$ mm ⟶ R.H.S = VV ? L.H.S = VV ✓

⋮

$r_3 =$